

Comments:

[1] This getting $G = -D^T$ (or vice versa) is tedious but it gives some advantages

Gresho's Book ← mathematical

[2] Saves space and work ΔU ΔP
in $e3xxx.f$ x60c [D;C] → stored in LHSP

all element
[2x2]
computations
take place
in these
files

2/20/09

[3] Later we will see that we use iterative solvers
⇒ matrix vector products

$$\vec{r} = A\vec{p}$$

$$\begin{bmatrix} K & G \\ D & C \end{bmatrix} \begin{bmatrix} \Delta U \\ \Delta P \end{bmatrix}$$

↳ this shape

How do we do that if we did not store G ?

→ Hardware the AP product to use $-D^T$

$$\underline{K} \underline{\Delta U} + \underline{G} \underline{\Delta P} = \underline{K} \underline{\Delta U} - \underline{D}^T \underline{\Delta P}$$

[4] Note the N_i^a , the i is just there to help us place the term in the 3×3 K_{ij}^{ab} matrix or D_j (1×3 matrix)

We use the same shape function regardless of the value of i . Furthermore, N_i^a is also the same for stabilize methods like we discussed before.

There are FEM methods, mixed methods, where the polynomial order of velocity is higher than pressure.

boxed {}

[5] It is fairly common when studying incompressible flow to study an auxiliary equation that is derived from the already discretized system,

$$\begin{bmatrix} \underline{\underline{K}} & \underline{\underline{G}} \\ \underline{\underline{G}}^T & \underline{\underline{C}} \end{bmatrix} \begin{bmatrix} \underline{\underline{\Delta U}} \\ \underline{\underline{\Delta P}} \end{bmatrix} = \begin{bmatrix} -\underline{\underline{R}} \\ \underline{\underline{r}} \end{bmatrix} \quad \text{Fully coupled momentum-continuity system}$$

FCMC

Extract momentum equation out

$$\underline{\underline{K}} \underline{\underline{\Delta U}} + \underline{\underline{G}} \underline{\underline{\Delta P}} = -\underline{\underline{R}}$$

imagine we know $\underline{\underline{K}}^{-1}$

$$\underline{\underline{\Delta U}} = \underline{\underline{K}}^{-1} [-\underline{\underline{R}} - \underline{\underline{G}} \underline{\underline{\Delta P}}] \rightarrow \text{plug } \underline{\underline{\Delta U}} \text{ into continuity eq.}$$

Second cont

$$-\underline{\underline{G}}^T \underline{\underline{\Delta U}} + \underline{\underline{C}} \underline{\underline{\Delta P}} = -\underline{\underline{r}}$$

$$-\underline{\underline{G}}^T \underline{\underline{K}}^{-1} [-\underline{\underline{R}} - \underline{\underline{G}} \underline{\underline{\Delta P}}] + \underline{\underline{C}} \underline{\underline{\Delta P}} = -\underline{\underline{r}} \quad \leftarrow \text{rearrange}$$

$$\underline{\underline{G}}^T \underline{\underline{K}}^{-1} \underline{\underline{G}} + \underline{\underline{C}} \underline{\underline{\Delta P}} = -\underline{\underline{r}} - \underline{\underline{G}}^T \underline{\underline{K}}^{-1} \underline{\underline{R}}$$

Discrete pressure poisson equation $(\nabla^2 P) = \nabla \cdot \text{mom}$

\rightarrow we don't work w/ this.

DPPE

Instead, we work with the DPPE

The incompressible code (PHASTA) calls a third party library that solves DPPE as a "preconditioner" to FCMC.

Why not just solve FCMC?

Eigenvalue spread between continuity equation (ΔP) and momentum equation (ΔU) so large that FCMC would have to be solved to very tight tolerances.

The "solution" solve DPPE (worse tolerance) then solve FCMC to loose tolerance but "constrained" to not violate DPPE solution

	1	2	3
1		2	3
2	4		5
3	7	8	

$$[6] \quad K_{ij}^{ab} = X k e b a \left(\overset{e}{\text{lines}}, \overset{ij}{1:9}, \overset{a}{\text{inshgl}}, \overset{b}{\text{inshgl}} \right)$$

nshgl = number of shape functions in the element
 i.e. tet, linear = 4

Assemble this fill sparse

$$xkete \text{ Assemble} \Rightarrow \text{lhs } k(1:nmb\text{-tot}, 9)$$

-i

the number of non-zeros in the matrix

$$XGOC \Rightarrow \text{lhs } P(1:nmb\text{-tot}, 4)$$

[7] Above we mentioned \underline{k}^{-1} . We can't really get this but instead we use $\underline{\underline{k}}^{-1} \approx \underline{\underline{k}}^{-1}$ typically $\underline{\underline{k}} = \underline{\underline{k}}^l$ lumped diagonal of the real \underline{k} .
 Trivial to invert.

Note: \exists something sort of in between compressible and incompressible flow.

\Rightarrow pseudo-compressible flow

only really sensible for acoustics

The idea is to put $\frac{\partial p}{\partial t}$ back into continuity equation

$$\frac{\partial p}{\partial t} = \frac{\partial p}{\partial p} \frac{\partial p}{\partial t}$$

$$\alpha \frac{\partial p}{\partial p} \frac{\partial p}{\partial t} + p u_{ii} = 0$$

$$\frac{\alpha}{p} \frac{\partial p}{\partial t} + u_{ii} = 0$$

Extra time term in continuity equation.

Probably push $\alpha f_p = \alpha E u$

$$\alpha m_p = \alpha m u$$

$$\gamma_p = \gamma_u$$

This technique is also pursued as a relaxation technique.

$$\frac{\partial p}{\partial t} \rightarrow \frac{\partial p}{\partial \tau} \quad \leftarrow \text{pseudo time}$$